ABSTRACT

To better understand the thermal effects from the engine exhaust jet flow on the thermo-mechanical behavior of E-2C aircraft structures, a unidirectional partitioned code fluid-thermal-structure coupling analysis was conducted by using the CFD solver of FLUENT and the CSD solver of ABAQUS connected via mesh-based parallel-code coupling interface (MpCCI). In the CFD analysis, a Multi-Reference Frame model was used to simulate the rotation of the engine propeller. The outer surface of E-2C aircraft was set as adiabatic walls. Steady thermal flow field of the aircraft with outer-wing folded and unfolded were both analyzed. The locations of the most seriously heated areas were predicted based on the CFD results. It showed that the lower parts of vertical tails were the most seriously heated parts when wings were folded, while the temperature of whole aircraft structures remained normal when wings were unfolded. Based on the temperature field predicted by CFD analysis, the thermal strain field of the most seriously heated structures was calculated by using CSD method with the temperature field transferred by MpCCI server. The maximum thermal strain in the vertical tail appeared at the boundary of leading edge made by composite materials with a value of 1710 microstrains, which suggested a special treatment for the structural enhancement and thermal protection.

1. INTRODUCTION

E-2C is the major carrier-based airborne early-warning (AEW) aircraft in service[1]. During the use of this type of aircrafts, it was found that the service life of the wing panel structures would be adversely affected by the heat from the engine tail pipes when wings were folded[2]. Prediction of seriously heated area locations and analysis of the structure thermal strain become interesting topics while few research works have devoted to them. Here an unidirectional fluid-thermal-structure coupling computation of E-2C aircraft have been successfully demonstrated.
A common approach for modeling the rotation of engine propeller utilizes a multi-reference frame (MRF) model in steady-state condition, and a sliding mesh model in transient-state. A basic review of such models is given in Ref. 3-4. ABAQUS 6.14-1 is adopted to analyze the heat transfer inside aircraft structure as well as the structure thermal strain. The heat in the flow field is transferred to the outside surface of aircraft structure by Mesh-based parallel-code coupling interface (MpCCI). The purpose of this work is to explore a numerical fluid-thermal-structure analysis procedure for a carrier-based airborne early-warning aircraft.

2. METHODOLOGY

2.1. Governing equations

Thermal flow around the aircraft obeys fluid flow physical conservation law including three basic conservations (continuity, momentum, and energy). The system of equations describing the thermal flow field properties regardless of gravity are written in a differential, Cartesian form.

Continuity equation:
\[ \frac{\partial \rho}{\partial t} + \nabla \cdot (\rho U) = 0 \]  \hspace{1cm} (1)

Momentum equation:
\[ \frac{\partial (\rho U)}{\partial t} + \nabla \cdot (\rho U U) = -\nabla p + \nabla \cdot (\tau) \]  \hspace{1cm} (2)

Energy equation:
\[ \frac{\partial (\rho E)}{\partial t} + \nabla \cdot (U (\rho E + p)) = \nabla \cdot (k \nabla T + (\tau : U)) \]  \hspace{1cm} (3)

where \( \rho \) is the fluid density; \( t \) is time; \( U \) is the fluid velocity; \( p \) is pressure; \( \tau \) is the viscous stress tensor; \( E \) is total energy including internal energy and kinetic energy; \( k \) is the fluid thermal conductivity; \( T \) is the fluid temperature.

To simulate the flow moving along with the propeller, these equations are formulated in terms of absolute velocity in a reference frame that can rotate around the origin with angular velocity. And the relative velocity, \( U_r \), is defined as
\[ U_r = U - \Omega \times R \]  \hspace{1cm} (4)

where \( \Omega \) is the angular velocity, \( R \) is the radius of the rotating mesh and the interface with the fixed mesh.

For a ideal gas, the equation of state is
\[ p = \rho RT \]  \hspace{1cm} (5)

In the case of turbulent flows, additional field equations such as the transport of turbulent kinetic energy and dissipation rate are required for enclosure. Here k-epsilon model is chosen as it gives reasonable good prediction between simulation results and experimental data. In short, k-epsilon model solves for turbulent kinetic energy, \( k \), and its rate of dissipation \( \epsilon \), which is coupled with turbulent viscosity \( \mu_t \). This model is given as:
\[
\frac{\partial (\rho k)}{\partial t} + \nabla \cdot (\rho V k) = \nabla \left( \mu + \frac{\mu_t}{\sigma_k} \right) \nabla k + G_k - \rho \varepsilon \quad (6)
\]
\[
\frac{\partial (\rho \varepsilon)}{\partial t} + \nabla \cdot (\rho V \varepsilon) = \nabla \left( \mu + \frac{\mu_t}{\sigma_\varepsilon} \right) \nabla \varepsilon + C_1 \frac{\varepsilon G_k}{k} + C_2 \frac{\varepsilon^2}{k} \quad (7)
\]

where \( G_k \) is the generation of turbulence kinetic energy due to mean velocity gradients; \( C_1 \) and \( C_2 \) are constants; \( \sigma_k \) and \( \sigma_\varepsilon \) are respectively turbulent Prandtl numbers for \( k \), and \( \varepsilon \); \( \mu_t \) is turbulent viscosity expressed as:

\[
\mu_t = C_\mu \rho k^2 / \varepsilon \quad (8)
\]

With a mean rate of strain tensor \( S_{i,j} \) taken into account, the renormalization group (RNG) k-epsilon model was finally chosen as it is more responsive to the effects of rapid strain and streamline curvature than the standard k-epsilon model\(^{[5]}\). And the values of the equation coefficients\(^{[6]}\) are

\[
\begin{align*}
C_\mu &= 0.0845; \quad C_1 = 1.42 - \frac{\eta(1 - \eta/\eta_0)}{1 + \beta \eta^3}; \quad C_2 = 1.68 \\
\sigma_k &= 0.7179; \quad \sigma_\varepsilon = 0.7179; \quad \tilde{\eta} = Sk / \varepsilon \\
S &= (2S_{i,j}S_{i,j})^{1/2}; \quad \eta_0 = 4.38; \quad \beta = 0.015
\end{align*}
\]

\[
S_{i,j} = \frac{1}{2} \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)
\]

### 2.2. Aerodynamic model

Dynamic mesh CFD for a propeller aircraft can become very expensive in terms of CPU time. Consider the three-dimensional models of E-2C aircraft, as shown in Fig.1. The diameter of the propeller is 4.14m, around which millions of mesh will be smoothed and computed in each time step.

MRF model, however, introduces a rotate velocity to the frame of rotating fluid zone around the aircraft propeller without mesh recomputation. This reduces the run time dramatically. The inlet, outlet, and wall boundary conditions are designed 15 times the length of the aircraft away from the aircraft to get a solution that is independent of the their exact position, and a symmetry of the model has been adopted.

Aircraft outside surface and the deck ground are set as adiabatic wall. Details about the model is illustrated in Fig.2. Two parts of fluid zone are meshed with unstructured mesh respectively and then assembled together. Interfaces are defined for the interpolation of fluxes and conserved variables from one side to the other. Fluid environment temperature is 300K with standard atmospheric pressure. T56-A-427 Engine made by Allison Engine Company is adopted by E-2C, the air mass flux of the engine is about 15.2kg/s\(^{[1]}\), and the temperature of the exhaust gas is set as 750K. Rotate speed of the propeller is set as 1000rpm.
2.3. Structural model

According to the CFD results, the outer vertical tail was found to be the seriously heated area and further structural thermal analysis was required. Therefore, a structure model of the vertical tail has been supplied, as shown in Fig.3. This model contained details about the lower vertical stabilizer including the leading edge, axle beams, ribs, outer skins and stiffeners, besides, the rudders and the rest of structures which remained normal temperature were simplified as solid part. Structures were primarily made by metal, here TC4 titanium alloy was chosen. However, the leading edges of the empennage were designed to be made by fiber reinforced composite materials for reducing the vertical tail resistance effect on the radar performance. The material parameters are shown in Tab.1.
ABAQUS 6.14-1 was utilized to analyze the heat transfer process through the vertical tail and the thermal strain of each component. Structures were tied together on contact surfaces, and a total of 62 tie constraints have been created. The initial temperature of the vertical tail was pre-defined as 300K. Solid elements, C3D8T, with both thermally coupling and trilinear capabilities, were used to discretize the whole structure model. The stacking sequence of the leading edge composite layers was defined as [-45°/0°/45°/90°].

<table>
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<th>Tab.1 Material parameters</th>
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<tr>
<td>CCF300 /5228A</td>
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</tr>
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<td>0.25</td>
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<tr>
<td>TC4</td>
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2.4. Fluid-thermal-structure coupling method
In general, there are two different approaches of code coupling[7]. Monolithic coupling approach solved both disciplines in one system of equations, without any interpolation at the interface of flow and structure but required identical discretizations for both domains. Partitioned code coupling however employs separate discretizations for each discipline but requires interpolation at the interface. Within this work, the CFD solver of FLUENT 15.0 is unidirectionally coupled to the CSD solver of ABAQUS 6.14.1 via MpCCI 4.4.1, as shown in Fig.4.
3. RESULTS

3.1. CFD Results and discussion

For prediction of seriously heated area locations on E-2C aircraft, steady thermal flow field computations with wings unfolded and folded were both performed. The CFD mesh was comprised of about 5200,000 tetrahedral cells computed using ICEM CFD. The flow was treated as compressible ideal gas.

Full Multigrid initialization (FMG initialization) was used at the start of calculation as it could provide a better initial solution and flow convergence for rotating machinery.
The CFD results showed a seriously heated area location at the lower vertical tail, especially at the composite leading edge when wings were folded, as seen in Fig. 5, the max temperature was about 70 degrees centigrade while the vertical tail remained a normal temperature of 30 degrees centigrade with wings unfolded. It could be found that an obvious heat up would occur on the vertical tail when wings had been folded. Besides, the thermal flows showed little affects on structures in other places of the aircraft where the temperature was below 33 degrees centigrade.

Deflections have been found in the wake of exhaust flow due to the Bernoulli Principle that an increase in the speed of a fluid occurred simultaneously with a decrease in pressure, as shown in Fig. 6. When wings were unfolded, the propeller race under the outer wing panel spread over a wide range of areas while it was limited in a narrow space between the engine and the fuselage on the other side. The flow which was closed to the fuselage gained a higher velocity. And a flow deflection away from the vertical tail appeared, making for little thermal effect on the structures. However, as the wings were folded, the situation changed and propeller race around the fold wings became a bit faster than the other side around the fuselage. Exhaust flow from the engine tail pipes, in this case, deflected away from the fuselage to the vertical tail, heating up structures. As a result, exhaust flow and propeller race interacted with each other, formed a high temperature flow with a clockwise rotation which is the same as the rotating direction of the propeller.

3.2. CSD Results and discussion

A maximum temperature of about 70 degrees centigrade was found to appear at the leading edge of the outer vertical tail which required further structural thermal analysis. MpCCI was used to transfer the wall temperature unidirectionally from CFD
results to the FE-code as boundary conditions of heat transfer analysis, as shown in Fig.7. Regardless of the complex heat convection between the structure and the fluid, here a more conservative result has been obtained.

A steady coupled temperature-displacement analysis was presented using the temperature data in the steady CFD results. Time period was set as 10 with a fixed increment of 0.1. The CSD results showed that the maximum principal strain in the vertical tails appears at the boundary of the composite leading edge, as seen in Fig.8, the value is about 1700 microstrains, while in other metal structures the maximum principal strain was only 784 microstrains. Periodic thermal strain on the leading edge which was made by composite materials would result in thermal fatigue and adversely affected the mechanical performance, besides, structures became apt to undergo chemical corrosion in the wet environment on sea. A structure enhancement or some special thermal protection such as Thermal barrier coatings (TBCs) was suggested.

4 CONCLUSIONS
This paper describes the demonstration of a unidirectionally coupled fluid-thermal-structure procedure using ABAQUS and Fluent to analyze the thermal effect of engine exhaust jet flow on the structures of E-2C carrier-based airborne early-warning aircraft. When wings are folded, the vertical tail is found to be the most seriously heated area, the maximum temperature value is 70 degrees centigrade. The maximum thermal strain in the vertical tails appears at the boundary of leading edge with a value of 1710
microstrains, which suggests some special treatment for structure enhancement and thermal protection. The results demonstrate that the ABAQUS/MpCCI/Fluent coupling method is capable of analyzing the thermal fluid of E-2C aircraft and its effects on aircraft structures.

REFERENCES


